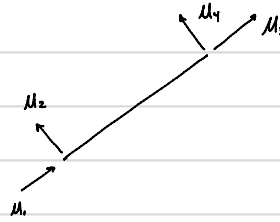


Uniaxial $M_1 \rightarrow \dots \rightarrow M_2$

2D Truss

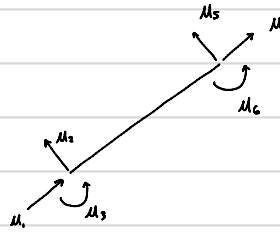


* Beam *

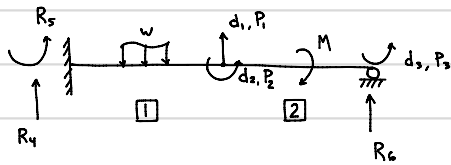


transverse displacements
and rotations

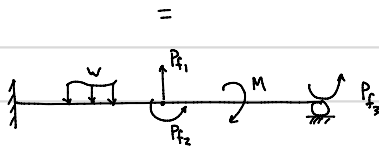
2D Frame
(later)



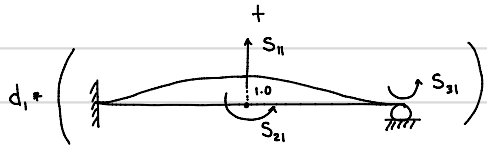
axial / transverse displacements
and rotations



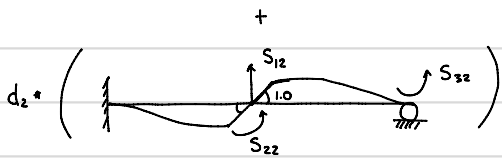
Structural-level Superposition



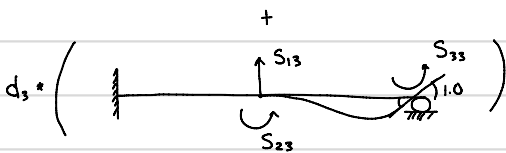
Apply loading $d_1 = d_2 = d_3 = 0$



$d_1 = 1.0, d_2 = d_3 = 0$



$d_2 = 1.0, d_1 = d_3 = 0$



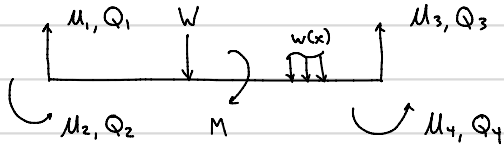
$d_3 = 1.0, d_1 = d_2 = 0$

$$\begin{aligned} P_1 &= P_{f1} + S_{11} d_1 + S_{12} d_2 + S_{13} d_3 \\ P_2 &= P_{f2} + S_{21} d_1 + S_{22} d_2 + S_{23} d_3 \\ P_3 &= P_{f3} + S_{31} d_1 + S_{32} d_2 + S_{33} d_3 \end{aligned}$$

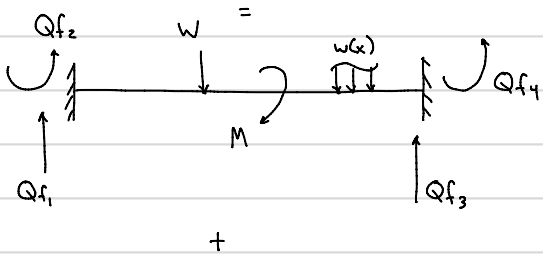
$$\begin{Bmatrix} P_1 \\ P_2 \\ P_3 \end{Bmatrix} = \begin{Bmatrix} P_{f1} \\ P_{f2} \\ P_{f3} \end{Bmatrix} + \begin{bmatrix} S_{11} & S_{12} & S_{13} \\ S_{21} & S_{22} & S_{23} \\ S_{31} & S_{32} & S_{33} \end{bmatrix} \begin{Bmatrix} d_1 \\ d_2 \\ d_3 \end{Bmatrix}$$

$$\{P - P_f\} = [S] \{d\}$$

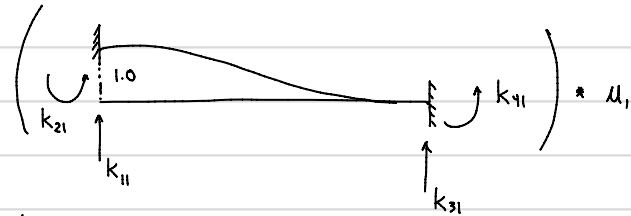
No difference in local/global coordinates for beams $\{Q\}, [k], \{u\}$



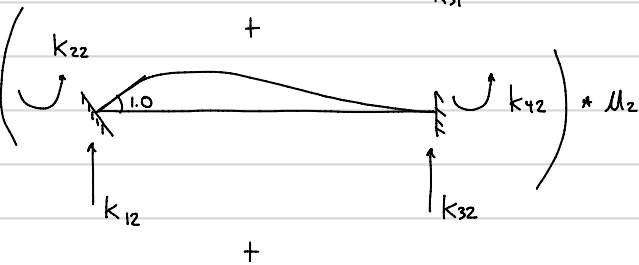
Member-level superposition



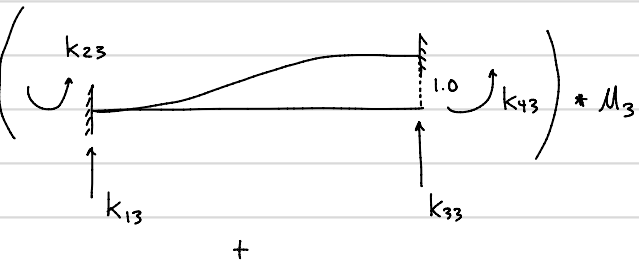
Apply loading $u_1 = u_2 = u_3 = u_4 = 0$



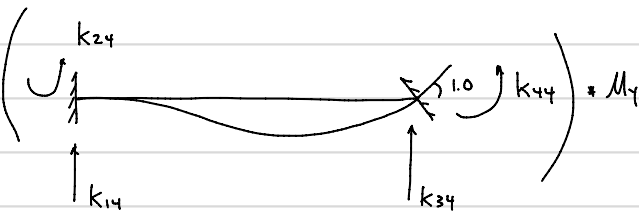
$u_1 = 1.0, u_2 = u_3 = u_4 = 0$



$u_2 = 1.0, u_1 = u_3 = u_4 = 0$



$u_3 = 1.0, u_1 = u_2 = u_4 = 0$



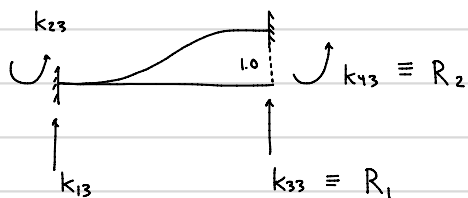
$u_4 = 1.0, u_1 = u_2 = u_3 = 0$

Now we write joint equilibrium equations

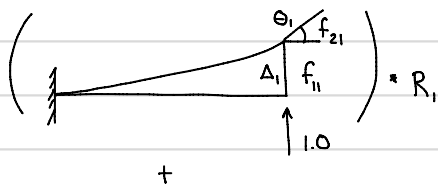
$$\begin{Bmatrix} Q_1 \\ Q_2 \\ Q_3 \\ Q_4 \end{Bmatrix} = \begin{Bmatrix} Q_{f1} \\ Q_{f2} \\ Q_{f3} \\ Q_{f4} \end{Bmatrix} + \begin{bmatrix} k_{11} & k_{12} & k_{13} & k_{14} \\ k_{21} & k_{22} & k_{23} & k_{24} \\ k_{31} & k_{32} & k_{33} & k_{34} \\ k_{41} & k_{42} & k_{43} & k_{44} \end{bmatrix} \begin{Bmatrix} M_1 \\ M_2 \\ M_3 \\ M_4 \end{Bmatrix} \quad \{Q\} = \{Q_f\} + [k]\{M\}$$

Need to derive stiffness terms

3rd column of $[k]$, $u_3 = 1.0$, $u_1 = u_2 = u_4 = 0$

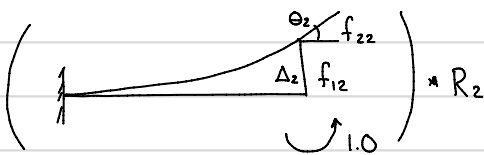


indeterminate (2nd degree) - use superposition!
2 redundants (flexibility approach)



$$R_1 = 1.0, R_2 = 0$$

Axial $P = \left(\frac{EA}{L}\right) \delta$
↑ displacement
↑ stiffness (k)



$$R_1 = 0, R_2 = 1.0$$

$$\delta = \left(\frac{L}{EA}\right) P$$

↑ flexibility $\frac{1}{k} \equiv f$

Compatibility

$$u_3 = \Delta_1 + \Delta_2 = 1.0$$

$$1.0 = f_{11} R_1 + f_{12} R_2$$

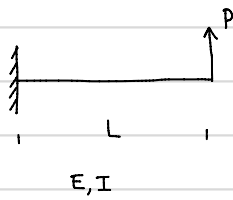
$$\begin{Bmatrix} 1.0 \\ 0 \end{Bmatrix} = \begin{bmatrix} f_{11} & f_{12} \\ f_{21} & f_{22} \end{bmatrix} \begin{Bmatrix} R_1 \\ R_2 \end{Bmatrix}$$

$$u_4 = \Theta_1 + \Theta_2 = 0$$

$$0 = f_{21} R_1 + f_{22} R_2$$

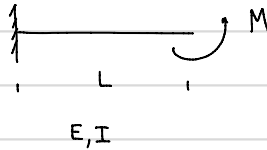
* Stiffness-based superposition: apply unit translations/rotations → equations from joint equilibrium
Flexibility-based superposition: apply unit forces/moments → equations from joint compatibility

We already know how to derive flexibilities (from moment-curvature)



$$\theta_1 = \frac{PL^2}{2EI}$$

$$\Delta_1 = \frac{PL^3}{3EI}$$



$$\theta_2 = \frac{ML}{EI}$$

$$\Delta_2 = \frac{ML^2}{2EI}$$

$$\Delta_1 = f_{11} P \quad f_{11} = \frac{L^3}{3EI}$$

$$\theta_1 = f_{21} P \quad f_{21} = \frac{L^2}{2EI}$$

$$\Delta_2 = f_{12} M \quad f_{12} = \frac{L^2}{2EI}$$

$$\theta_2 = f_{22} M \quad f_{22} = \frac{L}{EI}$$

$$\begin{Bmatrix} 1.0 \\ 0 \end{Bmatrix} = \begin{bmatrix} f_{11} & f_{12} \\ f_{21} & f_{22} \end{bmatrix} \begin{Bmatrix} R_1 \\ R_2 \end{Bmatrix} \quad [f] = \begin{bmatrix} \frac{L^3}{3EI} & \frac{L^2}{2EI} \\ \frac{L^2}{2EI} & \frac{L}{EI} \end{bmatrix}$$

Solving system of equations yields $R_1 = \frac{12EI}{L^3} = k_{33}$ $R_2 = \frac{-6EI}{L^2} = k_{43}$

From element equilibrium $\sum F_y = 0$ $k_{13} + k_{33} = 0$ $\therefore k_{13} = \frac{-12EI}{L^3}$

$\sum M_b = 0$ $k_{23} + k_{43} + k_{33} \cdot L = 0$ $\therefore k_{23} = \frac{-6EI}{L^2}$

Following the same superposition procedure for the 1st, 2nd, and 4th DOFs (i.e. columns)

$$[k] = \frac{EI}{L^3} \begin{bmatrix} 12 & 6L & -12 & 6L \\ 6L & 4L^2 & -6L & 2L^2 \\ -12 & -6L & 12 & -6L \\ 6L & 2L^2 & -6L & 4L^2 \end{bmatrix}$$